

4B.13 Tube Flow for the Truncated Power-Law Fluid

One way to estimate the importance of the zero-shear-rate viscosity on the volume flow rate in a tube is by using the simple truncated power-law model given in Table 4.5-1, Eqs. A and B. Let r_0 denote the radial position where $\dot{\gamma} = \dot{\gamma}_0$, that is where the viscosity changes from the zero-shear-rate value to the power-law function.

- How must r_0 be restricted in order to use the truncated power-law model? Show that this corresponds to $(\eta_0 \dot{\gamma}_0 / \tau_R) < 1$ where τ_R is the wall shear stress in the tube.
- Solve for the velocity field for $r \leq r_0$ and $r \geq r_0$. Match the results at $r = r_0$ in order to eliminate the last integration constant.
- Show that the volume flow rate is given by Eq. H in Table 4.5-2.
- Show that the result in part (c) reduces to the power-law result given in Eq. B of Table 4.2-1 or Eq. F of Table 4.5-2 when $(\eta_0 \dot{\gamma}_0 / \tau_R) \ll 1$. Note that the power-law consistency index m is related to the truncated power-law constants by $m = \eta_0 \dot{\gamma}_0^{1-n}$. Where does this come from?

4B.14 Variational Functional for Rod

Consider the functional in Eq. 4.3-16 and parallel the development in Eq. 4.3-3 *et seq.* In particular let $\bar{\Theta}(\xi) = \Theta(\xi) + \varepsilon \eta(\xi)$, where $\Theta(\xi)$ is the solution to Eqs. 4.3-13, 14 and 15, and $\eta(\xi)$ is an arbitrary function with $\eta(1) = 0$.

- Show that when terms through second order in ε are retained one finds:

$$J\{\bar{\Theta}(\xi)\} = J\{\Theta(\xi)\} + \varepsilon^2 J\{\eta(\xi)\} \quad (4B.14-1)$$

- Use the above to show that $\Theta(\xi)$ makes J a minimum.

4B.15 Development of Design Equation for Manifold of a “Coat-Hanger” Die (Power Law)²³

Plastic sheeting can be made by extruding the molten polymer through a “coat-hanger” die made up of an entry tube, two manifolds, and a slit (see Fig. 4B.15). The manifold is a tube of circular cross section, whose radius \bar{R} varies in the direction \bar{z} of the manifold axis. Our object is to design the manifold (i.e., find $\bar{R}(\bar{z})$) so that the flow through the slit will be uniform; that is the volume flow rate of the slit must not vary in the x -direction.

The slit has a total width $2W$ and has a thickness $2B$. The volume rate of flow into the entry tube is $2Q_0$, with half of the fluid going into the left manifold and half into the right manifold.

- Consider a width Δx of the slit. What is the volume flow rate through this portion of the slit?
- Make a mass balance over a length $\Delta \bar{z}$ of the manifold tube and then let $\Delta \bar{z}$ go to zero to get the differential equation:

$$-\frac{d\bar{Q}}{d\bar{z}} = \frac{Q_0}{W} \cos \alpha \quad (4B.15-1)$$

where \bar{Q} is the volume flow rate at \bar{z} . Draw a carefully labelled diagram to show how you derive this relation.

- Let $\bar{p}(\bar{z})$ be the pressure as a function of \bar{z} in the manifold. Let $p(z)$ be the pressure in the slit. Why is $p(-L(x)) \doteq \bar{p}(\bar{z})$? How are x and \bar{z} related? How are W and \bar{L} related?

²³ J. R. A. Pearson, *Trans. J. Plast. Inst.*, **32**, 239 (1964); J. R. A. Pearson, *Mechanics of Polymer Processing*, Elsevier Applied Science, New York (1985), §10.2. See also Z. Tadmor and C. G. Gogos, *Principles of Polymer Processing*, Wiley, New York (1979), pp. 545–551.

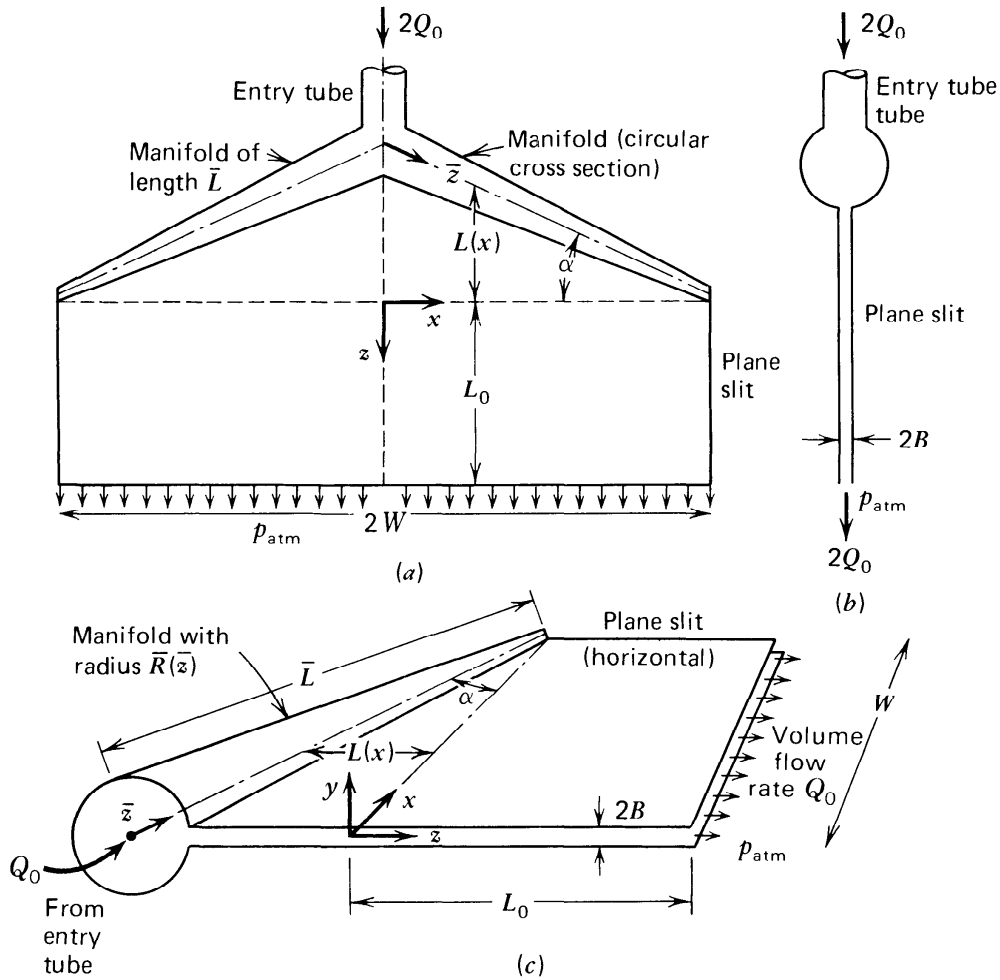


FIGURE 4B.15. The “coat-hanger” die, so named because the entry tube and manifold resemble a coat hanger. (a) Top view; (b) side view (looking in x -direction at a cut through the apparatus in the yz -plane); (c) one manifold and half of slit.

d. Adapt the power-law-fluid plane-slit-flow formula to a region of width Δx . Then use the results in (c) to obtain, finally,

$$-\frac{d\bar{p}}{d\bar{z}} = \left[\frac{Q_0}{W} \frac{[(1/n) + 2]}{2B^2} \right]^n \frac{m}{B} \sin \alpha \quad (4B.15-2)$$

e. Write the power-law analog of the Hagen–Poiseuille formula locally over a segment $d\bar{z}$ of the tube. This should contain $d\bar{p}/d\bar{z}$.

f. Combine the results of (b), (d), and (e) to get a differential equation for \bar{R} as a function of \bar{z} . This should be of the form

$$\bar{R}^{(1/n)+2} d\bar{R}/d\bar{z} = \text{const} \quad (4B.15-3)$$

g. Integrate the differential equation to get $\bar{R}(\bar{z})$, the shape of the manifold. Take \bar{R} to be zero at $\bar{z} = \bar{L}$ (just to make the problem a little simpler). Find ultimately that the desired design formula is

$$[\bar{R}(\bar{z})]^{3n+1} = \left[\frac{2}{\pi} \frac{[(1/n) + 3]}{[(1/n) + 2]} \right]^n \frac{2B^{2n+1}}{\sin \alpha} (W - x)^n$$

Show that if $n = 1/2$, then $\bar{R} \propto (W - x)^{1/5}$.

4B.15 Development of Design Equation for Manifold of a "Coat Hanger" Die (PLF) [RM]

a) For uniform flow: $\Delta Q = \frac{Q_0}{W} \Delta X$

b) $\bar{Q} :=$ Flow rate into a section at a distance \bar{z}

Flow rate out of manifold section at $(\bar{z} + \Delta \bar{z})$
 $= \bar{Q} + \Delta \bar{Q}$

$$\left\{ \begin{array}{l} \text{Flow into} \\ \text{manifold} \end{array} \right\}_{\bar{z}} = \left\{ \begin{array}{l} \text{Flow out} \\ \text{of manifold} \end{array} \right\}_{\bar{z} + \Delta \bar{z}} + \left\{ \begin{array}{l} \text{Flow into} \\ \text{plane slit} \end{array} \right\}_{\bar{z} + \Delta \bar{z}}$$

$$\bar{Q} = \bar{Q} + \Delta \bar{Q} + \frac{Q_0}{W} \Delta \bar{z} \cos \alpha$$

$$\therefore -\frac{\Delta \bar{Q}}{\Delta \bar{z}} = \frac{Q_0}{W} \cos \alpha \rightarrow -\frac{d\bar{Q}}{d\bar{z}} = \frac{Q_0}{W} \cos \alpha$$

c) The pt. $z = -L(x)$ is the pt. \bar{z} . Thus, if we denote the pressure distribution in the manifold by $\bar{p}(z)$ and that along the slit by $p(z)$:

$$p(-L(x)) = \bar{p}(\bar{z}); \quad \bar{z} = x / \cos \alpha; \quad \bar{L} = W / \cos \alpha$$

d) From Table 4.2-1, The pressure drop along plane

$$\text{slit: } -\frac{dp}{dz} = \left[\frac{Q_0 [\frac{1}{n} + 2]}{2B^2 W} \right]^{\frac{n}{B}}$$

4B.15

We assume that the pressure is uniform all across the slit. Therefore, the pressure drop along a distance L' (along \bar{z}) of the manifold is equal to that along a distance $L' \sin \alpha$ (along z) of the slit.

$$dz = d\bar{z} (\sin \alpha)$$

$$\therefore -\frac{d\bar{p}}{d\bar{z}} = \left[\frac{Q_0 [\frac{1}{n}+2]}{W 2B^2} \right]^n \frac{m}{B} \sin \alpha$$

e) From Ex. 4.2-1, we can write the equivalent of the Hagen-Poiseuille eqn. for the flow of a power-law fluid:

$$\bar{Q} = \frac{\pi \bar{R}^3}{\frac{1}{n}+3} \left[-\frac{d\bar{p}}{d\bar{z}} \frac{\bar{R}}{2m} \right]^{1/n} \quad \text{OR} \quad -\frac{d\bar{p}}{d\bar{z}} = \left[\frac{\bar{Q} [\frac{1}{n}+3]}{\pi \bar{R}^3} \right]^n \frac{2m}{\bar{R}}$$

$$f) \frac{d\bar{Q}}{d\bar{z}} = \pi \left[-\frac{d\bar{p}}{d\bar{z}} \cdot \frac{1}{2m} \right]^{1/n} \bar{R}^{\frac{1}{n}+2} \frac{d\bar{R}}{d\bar{z}} \quad \text{by diff. above eqn.}$$

Subst. $\frac{d\bar{p}}{d\bar{z}}$ from e) and $\frac{d\bar{Q}}{d\bar{z}}$ from b) into above eqn:

$$\bar{R}^{\frac{1}{n}+2} \frac{d\bar{R}}{d\bar{z}} = -\frac{Q_0}{\pi W} \cos \alpha \left[\frac{Q_0^n (\frac{1}{n}+2)^n}{(2B^2 W)^n} \cdot \frac{m \sin \alpha}{B} \cdot \frac{1}{2m} \right]^{-1/n} \quad \text{or}$$

$$\bar{R}^{\frac{1}{n}+2} \frac{d\bar{R}}{d\bar{z}} = -\frac{2B^2 \cos \alpha}{\pi (\frac{1}{n}+2)} \left[\frac{2B}{\sin \alpha} \right]^{1/n} = \text{constant}$$